

AIN SHAMS UNIVERSITY FACULTY OF ENGINEERING DEPARTMENT OF AUTOMOTIVE ENGINEERING

BASES OF AUTOMATED ENGAGEMENT OF DRY FRICTION CLUTCH IN MOTOR VEHICLES

A THESIS SUBMITTED IN PARTIAL FULFILMENT OF THE REQUIREMENTS FOR THE DEGREE OF MASTER OF SCIENCE IN MECHANICAL ENGINEERING

 $\mathbf{B}\mathbf{Y}$

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STATEMENT

This thesis is submitted to Ain-Shams University for the degree of: Master in mechanical engineering.

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No part of this thesis has been submitted for a degree or a qualification at any other University or Institute.

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FOR MY BELOVED FAMILY
WHO ALWAYS ENCOURAGES ME AND
OVERSHADOWS ME WITH THEIR
CONTINUOUS LOVE AND CARE.

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SUMMARY

The dry friction clutch plays important functions in motor vehicles during the movement of stationary vehicles, and ensures easy gear shifts during the speeding up and slowing down motion. On the other hand, the driver skill and behaviour in operating the clutch during engagement and disengagement has unquestionable effect on the achievement of the desirable performance and on the service life of the clutch. Further, the service life has direct relation to the high repair costs, and down time needed for the early damaged clutch. Moreover, the optimum operation of the clutch has a great impact on the fuel savings.

An apparatus was designed and built to record the clutch pedal and the accelerator displacements during the disengagement and engagement of the clutch at the different speed shifts of the vehicle.

Tests were carried out on eight drivers of different skill levels, using the same test vehicle, and under the same operating conditions. Results of the tests were recorded and used to deduce the clutch slip time, and the amount of the generated heat as a result of that slip. Comparisons between the drivers' performance have been carried out.

NOMENCLATURE

SYMBOL	MEANING	UNITS
Af	Frontal area of the vehicle.	[m²]
f _r	Tyre rolling resistance coefficient.	
Fa	Air resistance to the vehicle motion.	[N]
Fd	Draw-bar pull of the trailer.	· [N]
Fg	Grade resistance to the vehicle.	נא ז
Fi	Inertial resistance of the vehicle.	[N]
Fr	Rolling resistance at wheels.	[N]
FR	Total resistance force acting on wheels.	[N]
g	Gravitational acceleration constant, 9.81	[m/s ²]
ig	Gear box ratio.	
io	Final drive reduction ratio.	
I ₁	Equivalent mass moment of inertia of the en	gine
	moving parts.	[kg m²]
sI	Equivalent mass moment of inertia of the ve	hicle,
	referenced at the speed of the clutch shaft	. [kg m²]
I eq	Equivalent mass moment of inertia of the ve	hicle,
	seen at the driving wheels' axle.	[kg m²]
ı w	Inertia moment of the wheels.	[kg m²]
k _a	Air resistance coefficient.	[N.S ² /m ⁴]
М	Mass of the vehicle.	[kg]
n,n2	Indices of engine and clutch torque function	s chosen
	generally as: 0,0.2,0.4,0.6,0.8,1,1.2,1.4,2	,3,4,5
rd,r	Dynamic & rolling radii of the driving whee	ls. [m]

TC	Maximum clutch torque.	[Nm]
TE	Maximum engine torque.	[Nm3
Tc(t)	Clutch torque function.	
Te(t)	Engine torque function.	
TrCtO	Total resisting torque on wheels, referenced a	t
	the clutch output shaft, function of time.	[Nm]
Tw	Resultant torque on wheels.	[Nm]
t	Time.	· [s]
W	Weight of the vehicle.	[N]
×	Linear translation of the wheels.	[m]
×	Linear acceleration of the wheels.	[m/s²]
α	Road slope angle.	[°]
6 ₁	Relative delay time of accelerator pedal movem	ent.[s]
క్ష	Relative delay time of clutch pedal movement.	[s]
γ	Ratio of engine maximum torque, [0 < γ < 1].	
η	Mechanical efficiency of the vehicle transmiss	ion.
^τ 1, ^τ 3	Build up time of the engine torque.	[2]
ε ^τ	Build up time of the clutch torque.	[s]
Ω ₁ ,Ω ₂	Initial angular velocities of the engine and t	he.
	friction plate (s), respectively.	[r.p.m]
ω ₁ ,ω ₂	Instantanious angular velocities of the engine	and
	friction plate (s), respectively.	[r.p.m]
Ø	Angular displacement of driving wheels.	[rad]
8	Angular acceleration of driving wheels.	rad/s ²]

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