# STUDY AND DESIGN OF AN ECONOMIC FLAT PLATE SOLAR HEATER IN EGYPT

M. Sc. THESIS

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Praise be to God,

Lord of the World,

by whose grace this work

has been completed.

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- ARABIC SULMARY.

# SUMMARY

### SUMMARY

This work presents a theoretical and an experimental investigation for the thermal performance of two systems of solar collector, namely:

- 1- The traditional collectors: Three types has been developed and investigated with different shaped tubes such as:

  Type I of rectangular serpentine tubes, type II of circular serpentine tubes and type III of separate tubes.
- 2- The compact collectors: Three types with different form of design have been developed and tested such as: steel box, gypsum basin and steel basin.

The collectors were tested under typical weather conditions of Cairo, Egypt at the field of tests at the Building Research Center in Dokki.

Computer programs have been developed for each type and for each collector. The data used to run the programs were the same as recorded during the measurments. The programs have been used to investigate some construction factors to improve the thermal performance of the collectors.

The theoretical approach deals with the estimation of the incident solar radiation on the tilted surfaces, the overall heat loss coefficient, the fluid heat transfer coefficient, temperature of plate, cover and fluid and the efficiency which is given by a set of equations.

Generally, the outlet water temperature reaches to 78°C for the intermittent discharge per half hour interval, 55°C for the continuous discharge of 0.004 kg/m $^2$ s, 52°C for the discharge at the end of the day and 39°C for the discharge of 10 kg/m $^2$  per hour or for the continuous discharge 0.004 kg/m $^2$ s from the storage.

An economical study is also included to indicate the cost of each collector and to compare between them. The cost reaches to 43 L.E. for the steel basin which gives 44.8 kg at the end of the day (i.e. at 1330 hour) with outlet water temperature 52°C.

The study also introduced the compact system acts as a solar collector and storage which has a low cost, simple technology, good thermal performance and is suitable for rural areas.

## NOMENCLATURE

$^{ m A}_{m c}$	:	collector area.	<b>m</b> 2
ao,	а, є	, b: Coefficient in empirical r	elationships non
$^{\mathrm{C}}_{\mathrm{A}}$	:	cost per unit of collector area.	L •E
Ср	:	bond conductance.	₩/m²°C
$\mathtt{C}_{\mathrm{E}}$	:	cost of equipment.	L.B.
g	:	specific heat.	KJ/Kg°C
${\tt c}_{\bf s}$	:	total cost of equipment.	L.B
D	:	diameter damping factor	II.
d	:	market discount rate.	non
е	:	emissive power (base of natural l	ogarithm) non
<u> </u>	:	fin efficiency factor.	non
F	:	collector efficiency factor.	non
$\mathbb{F}_{II}$	:	collector flow factor.	non
$F_{ m R}$	:	collector heat removal factor.	non
Ë	:	gravitational acceleration.	m/sec <sup>2</sup>
Gr	:	grashof number.	non
G <sub>sc</sub>	:	solar constant	₩/ <b>m</b> <sup>2</sup>
h	:	heat transfer coefficient between	the fluid
·		and the wall.	₩/m <sup>2</sup> °C
h <sub>p-c</sub>	: :	heat transfer coefficient by conv	ection between
		plate and cover.	₩/m²°C
h <sub>p-w</sub>	, :	heat transfer coefficient by conv	_
		plate and water.	W/m <sup>2</sup> °C

W/m<sup>2</sup>°C

$\mathbf{E}_{\mathbf{r}}$	:	heat transfer coefficient by radiation. W/m	2°C
hrc-	ı <b>:</b>	heat transfer coefficient by radiation between W/m	2°c
		cover and air.	
hrp-	c <b>:</b>	heat transfer coefficient by radiation between W/m	2° c
		plate and cover.	
$\mathbf{L}_{\mathbf{w}}$	:	heat transfer coefficient by convection between W/m	2°c
		plate and water.	
h <sub>w-c</sub>	:	heat transfer coefficient by convection between W/m	<sup>2°</sup> c
		water and cover.	
I	:	is the incident radiation on a horizontal plane W/	2
Ib	:	is the hourly incident beam radiation on a hori- W/	m 2
		zontal plane.	
$\mathbf{I}_{\tilde{\mathbf{d}}}$	:	is the hourly incident diffuse radiation on a	n <sup>2</sup>
		horizontal plane.	
Ii	:	is an unpolarized incident radiation in a medium. W.	/m <sup>2</sup>
Ion	:	is the normal incident extraterrestrial radiat-	/m <sup>2</sup>
		ion on the plane on the n the day of the year.	
Ir	:	is the unporalized reflected radiation from a	/m <sup>2</sup>
_		medium.	
Ιդ	:	is the incident radiation on a tilted plane. W	/m <sup>2</sup>
-			
i	:		n -1
K	•	is the extinction coefficient.	_Y _Y
k	:	is the thermal conductivity, W.	/m <sup>2°</sup> 0

L thickness. ш 7 length. mean, mortgage interst rate, constant non m mass flow rate. Kg/m<sup>2</sup>s m medium, day of the year, constant in equations non number of covers, term of mortgage or economic N analysis. Nu Nusselt number. non Nusselt number for water fluid in tubes or boxes non Nue Prandth number. ₽,, : non present worth factor.  $P_{wF}$ non :  $P_{wN}$ present worth of the payment. non : energy per unit time. w/m<sup>2</sup> Q : total solar energy absorbed by collector. W/m2 Q W/m<sup>2</sup> rate of energy losses.  $Q_{\mathsf{T}_{-}}$ : W/m2 rate of energy storage. Q.S : rate of useful energy gain.  $Q_{11}$ Laduis, ratio. יי m, non Ra leigh number non  $R\alpha$ ratio of beam radiation on tilted plane to that  $R_h$ on horizontal plane. non Renold number. non Rو the perpendicular component of reflection of regr unpolarized radiation. non

rpan	r :	the parallel component of reflection of un-	non
		polarized radiation.	
r	:	the total reflection of unpolarized radiation	_
s	:	absorbed solar energy per unit area.	W/m²
T	:	temperature.	70
Ta	:	ambient temperature.	°C
$^{\mathrm{T}}$ c	:	cover temperature.	ەر
${f r}_{f f}$	:	fluid temperature.	°C
T fji	:	inlet fluid temperature.	°C
T <sub>fm</sub>	:	mean fluid temperature.	°C
T <sub>L</sub> o		outlet fluid temperature.	°C
gT	:	plate temperature.	ه د
	:	mean plate temperature.	۵۲
T <sub>pm</sub> T <sub>s</sub>	:	storage temperature.	°C
T sky		sky temperature.	~(
$\mathbf{T}_{W}$	:	water temperature.	°C
U <sub>b</sub>	:	back loss coefficient.	M/w <sub>5°</sub> C
ΰ <sub>e</sub>	:	edge loss coefficient.	W/m2°C
$\mathtt{U}^{\mathbf{L}}$	:	overall loss coefficient.	M\m 50 C
Us	•	storage loss coefficient.	W/m <sup>2</sup> °C
$\mathtt{U}_{\mathbf{t}}$	:	top loss coefficient.	₩/m²°C
4	:	wind speed.	m/s
W	:	distance between tubes.	m

Gre	ek:		
~	:	absorptance, thermal diffusivity	m <sup>2</sup> /s
В	:	glop <b>e</b> ~	degree
Ý	:	surface azimuth angle.	degree
8	:	declination	<b>negr</b> ee
6	:	emittance.	non
2	•	efficiency.	non
÷	:	angle between surface normal and incident radiation.	degree
بر	:	Kinametic viscosity	m <sup>2</sup> / <sub>E</sub>
T.		<b>V</b> iscosity	
P	:	reflectance.	K <b>_/ms</b> non
Pa	:	diffuse reflection	non
Vy Pa 67	:	Stefan - Boltzman constant.	/in < 1.4
Z	:	transmittance.	non
La	:	transmittance by absorption	non
Zb	:	beam transmition.	non non
Zd	:	diffuse transmition.	non
7per	:	transmittance perpendicular component.	non
Tr	:	transmition by reflection.	non
Ø	:	latitude angle.	aegree
$\omega$ :	:	hour angle.	degree

## Abbreviations:

C	:	calculate <b>d</b>
M	:	measured.
T <sub>f,t</sub>	:	hourly fluid temperature.
$\overline{\mathtt{T}_{\mathtt{p}}}$	:	daily mean plate temperature.
$t$ , $q^T$	:	hourly mean plate temperature.