DESIGN, PERFORMANCE ANALYSIS, AND MANUFACTURING OF PARABOLIC TROUGH SOLAR COLLECTOR USED FOR THERMAL LOADS

 $\mathbf{B}\mathbf{y}$

Eng. Mohamed Rashed Helmy AbdelMagid

A Thesis Submitted to the Faculty of Engineering at Cairo University in Partial Fulfillment of the Requirements for the Degree of MASTER OF SCIENCE

In

MECHANICAL POWER ENGINEERING

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FACULTY OF ENGINEERING, CAIRO UNIVERSITY GIZA, EGYPT 2015

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Thesis Title: DESIGN, PERFORMANCE ANALYSIS, AND MANUFACTURING OF PARABOLIC TROUGH SOLAR COLLECTOR USED FOR THERMAL LOADS.

Keywords: Parabolic trough collector, Intercept factor, Optical efficiency, Shading factor, End loss factor.

Summary: This thesis presents a detailed study on Parabolic Trough Solar Collectors applied with a water desalination unit using a multi-effect distillation technique. Thermal design procedures were implemented to determine the full dimensions of the collector. Then, optical and thermal losses from the collector were investigated taking into consideration the shading effect and the end loss factor. Also, transient thermal performance of the collector was studied throughout the annual seasons. It was found that the best performance will be achieved in May during which the desalination unit can be solar operated for seven hours daily, producing 15 m³/day of distillate water. Finally, the construction, manufacturing, and assembly steps of the parabolic trough collector were introduced.



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Nomenclature

Symbol	Quantity
$A_{aperture}$	Aperture Area, m ²
a	Accommodation Coefficient
b	Interaction Coefficient
c	Specific Heat, kJ/kg.K
C	Concentration Ratio
D	Receiver Diameter
d_D	Distance between the two units
d_r	Displacement of the receiver from the focus
d*	Universal non-random error due to receiver dislocation
f	Focal Length
f	Friction Factor
h	Heat Transfer Coefficient, W/m ² .K
k	Thermal Conductivity, W/m.K
$k(\theta)$	Incidence Angle Modifier
L	Collector Length, m
1	Un-illuminated Length of the receiver, m
l_{win}	Illuminated length of the second receiver, m
m	Oil mass, kg
n	Day number of the year
Nu	Nusselt Number
Pr	Prandtl number
$\dot{Q}_{ m absorbed}$	Solar irradiance absorbed by the receiver tube, W/m ²
$\dot{Q}_{heatloss}$	Solar irradiance loss by the receiver tube, W/m ²
$\dot{Q}_{ m Load}$	The steam generator load, W
$\dot{Q}_{ m useful}$	The useful energy carried by the heat transfer fluid, W
q' _{12conv}	Heat flux by convection from the receiver to the HTF, W/m

q'_{23cond} Heat flux by conduction in the receiver tube, W/m

q'_{3SolAbs} Heat flux absorbed by the receiver tube, W/m

q'_{34conv} Heat flux by convection from the receiver to the glazing, W/m

q'_{34rad} Heat flux by radiation from the receiver to the glazing, W/m

q'_{45cond} Heat flux by conduction in the glass glazing, W/m

q'_{5SolAbs} Heat flux absorbed by the glass glazing, W/m

q'_{56conv} Heat flux by convection from the glazing to ambient, W/m

q'_{57rad} Heat flux by radiation from the glazing to the sky, W/m

q'cond,bracket Heat flux loss in the supporting brackets, W/m

q'heatloss Heat flux loss from the receiver tube, W/m

Ra Rayleigh Number

Re Reynolds Number

t Time, hrs

T Temperature, K

T₆ Ambient Temperature, K

T₇ Effective Sky Temperature, K

U Overall Heat Transfer Coefficient, W/m².K

W Total Aperture Width, m

W_{eff} Un-shaded Aperture Width, m

X Shading Width, m

Greek Letters

 $\Phi_{\rm r}$ Rim Angle

Φ Latitude angle

б Stefan-Boltzmann constant

σ Standard deviation of random errors

 σ^* Universal parameter for random errors

β Tracking angle error

β* Universal parameter for tracking error

γ Intercept Factor

θ Incidence Angle

δ	Declination angle
ω	Hour angle
γ_{s}	Solar azimuth angle
Ψ	Orientation angle
ζ	Tracking angle
ρ	Mirror reflectivity
τ	Glazing transmisivity
α	Receiver absorbtivity
η_{o}	Optical efficiency

Emissivity

ε