

Ain Shams University

Faculty of Engineering

Position Control of Pneumatic Actuators Using Pulse Width Modulation Technique

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By:

Sherif Adel Mohamed Abdallah Elbelady

Supervised By:

Prof. Abdulaziz Morgan Abdulaziz

Dr. Hany El said Fawaz

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By

Sherif Adel Mohamed Abdallah Elbelady

B.Sc. Mechanical Engineering
Military Technical College

EXAMINERS COMMITTEE

Name	signature
Prof. Mohamed Abo Elella Abdellatif	••••••
Prof. Taher Gamal-Eldeen Abu-El-Yazied	•••••
Prof. Abdulaziz Morgan Abdulaziz	••••••
	Date://

STATEMENT

This thesis is submitted to Ain Shams University in partial fulfillment of the requirement for the M.Sc. Degree in Mechanical power engineering. The work included in this thesis has been carried out by the author in mechanical power engineering department, Ain Shams University. No part of the thesis has been submitted for a degree or a qualification at other university or institute.

Signature:

Name : Sherif Adel Mohamed Abdallah Elbelady

Date : /1/2017

Research Data

Name of Researcher : Sherif Adel Mohamed Abdallah Elbelady

Date of Birth : July, 26th 1978

Place of Birth : Egypt

Nationality : Egyptian

University degree : B.Sc. Mechanical Engineering

Department : Mechanical Power Department

University : Military Technical College

Date of Degree : June 2001

Current Job : Engineering Officer in Egyptian Forces

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ABSTRACT

Experimental and theoretical study is carried out to control the position of a pneumatic cylinder using online Self-Tuning PID controller (denoted as STNPID controller) based on neural network with back propagation scheme.

The inputs to the STNPID are the tracking error, set point and control action, while the outputs of the neural network are the optimized gains k_p , k_i and k_d .

The STNPID uses neural networks for online tuning of the PID controller gains depending on the environmental and system requirements in order to make the nonlinear system unaffected by the unpredictability of system's parameters and disturbances such as noise.

High speed digital control valves (operating at frequencies up to 150 Hz) are implemented to accurately control the position of the cylinder piston using Pulse Width Modulation (denoted as PWM) technique.

Results show that the STNPID controller is able to track both constant and variable set point trajectories efficiently by the pneumatic actuator system.

Comparison of the results of STNPID with the conventional PID controller shows that the self-tuning of the PID gains can cope with the nonlinearity of the pneumatic system which is incompatible with the linear behavior of the conventional PID controller. The tracking response for STNPID controller is enhanced with less overshoot and less steady state errors compared to the conventional PID. For square wave, the maximum overshoot is improved from 18.83 % for PID to 2.67 % for STNPID. The steady state error is also improved from 9.42 % to 2.67 %.

The oscillation behavior for the sinusoidal response of the conventional PID controller is damped by the STNPID controller and better response is observed. Also, the actuator acts smoothly and uniformly compared to the conventional PID controller with less oscillatory actuator response.

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LIST OF SYMBOLS

A	Cylinder area
Acr	Right side cylinder area
A_{cl}	Left cylinder area
$\mathbf{A_s}$	Stem area
a	Acceleration
$\mathbf{a}_{\mathbf{v}}$	Valve area
c	Sonic speed
$\mathbf{c_1}$	Speed of sound in air at nozzle inlet
$\mathbf{c_2}$	Speed of sound in air at nozzle throat
C_d	Coefficient of discharge
$C_{\mathbf{p}}$	Specific heat at constant pressure
$\mathbf{C_v}$	Specific heat at constant volume
d	Exact differentials
$\mathbf{E}_{\mathbf{sys}}$	System energy
e	Error
e _{rms}	Root mean square error
\mathbf{F}	Force
$\mathbf{F}_{\mathbf{e}}$	Equivalent force
Fext	External forces
f	Sigmoid function
$\mathbf{F_f}$	Static friction force
$\mathbf{F}_{\mathbf{v}}$	Coefficient of viscous friction
H	Enthalpy
h	Step size
$\mathbf{h_{i}}$	Inlet specific enthalpy
$\mathbf{h_o}$	Outlet specific enthalpy
T	Cost function

Derivative gain $\mathbf{k_d}$ $\mathbf{k}_{\mathbf{i}}$ Integral gain $\mathbf{k}_{\mathbf{p}}$ Proportional gain Runge kutta right pressure coefficient $\mathbf{k}_{\mathbf{pr}}$ $\mathbf{k}_{\mathbf{pl}}$ Runge kutta left pressure coefficient $\mathbf{k}_{\mathbf{Tr}}$ Runge kutta right temperature coefficient Runge kutta left temperature coefficient **k**TI Runge kutta velocity coefficient $\mathbf{k}_{\mathbf{v}}$ $\mathbf{k}_{\mathbf{v}}$ Runge kutta displacement coefficient L Work distance Mach number M Mass m Mass of the gas $\mathbf{m}_{\mathbf{g}}$ Piston mass. $\mathbf{m}_{\mathbf{p}}$ Left valve mass flow rate m_{vl} Right valve mass flow rate m_{vr} \mathbf{N} Number of wave cycles Input given by STNPID controller net_x Neuron output 0 Inlet flow pressure. \mathbf{p}_1 Outlet flow pressure \mathbf{p}_2 Patm Atmospheric pressure $\mathbf{P_b}$ Back pressure P_c Cylinder pressure Left side cylinder pressure $\mathbf{P}_{\mathbf{cl}}$ P_{cr} Right side cylinder pressure

 P_{crt}

 P_d

Critical pressure

Downstream pressure

P_s Supply pressure

P_{th} Throat pressure

P_u Upstream pressure

Q Heat

r Three inputs matrix

R Gas constant

t Time

T Air temperature

T₁ Inlet flow temperature

T₂ Outlet flow temperature

T_b Back temperature

T_{cl} Left cylinder temperature

T_{cr} Right cylinder temperature

T_{crt} Critical temperature

T_d Derivative time in seconds

T_i Integral time in seconds

T_{th} Throat temperature

T_{ur} Right upstream temperature

 T_{ul} Left upstream temperature

U Internal energy

u(**t**) Control action

u₁ Inlet flow velocity

u₂ Outlet flow velocity

V Volume

v_p Piston velocity

 $\Delta \mathbf{w}$ Increment of the connection weight

- $\mathbf{w_{ji}}$ Connection weights from the input neuron i at the input layer to the neuron at the hidden layer
- $\mathbf{w_{kj}}$ Connection weights from the hidden neuron j at the input layer to the neuron at the output layer
- y_a Actual value
- y_c Actual actuator position
- y_d Desired value
- y_{max} Maximum position

GREEK SYMBOLS

- **α** Momentum term
- γ Ratio of specific heats
- **δ** Back propagating error
- η Learning rate
- **θ** Bias
- v Mean fluid velocity
- ν₁ Inlet velocity
- v₂ Output velocity
- ρ_1 Upstream density
- ρ_2 Downstream density
- \sum Sigma